

(I) A reversible cycle is considered as shown in Fig. 1. The working fluid of mass m is an ideal gas whose specific heat is independent of temperature. From State 1 to State 2, the gas is compressed isothermally at a temperature T_L . From State 2 to State 3, the gas is heated at a constant pressure p_H . From State 3 to State 4, the gas expands isothermally at a temperature T_H . From State 4 to State 1, the gas is cooled at a constant pressure p_L . The gas constant and the specific-heat ratio of the gas are R and κ , respectively. Answer the following questions using the above given symbols of physical quantities. (25points)

- (1) Draw this cycle on a $T-S$ diagram with the numbers of States 1, 2, 3 and 4. (T : temperature, S : entropy)
- (2) Find the heat Q_{12} rejected from the gas during the process 1→2.
- (3) Find the heat Q_{23} added to the gas during the process 2→3.
- (4) Prove the relation $Q_{23} = Q_{41}$. Here, Q_{41} is the heat rejected from the gas during the process 4→1.
- (5) Find the net work W done by the gas during one cycle.
- (6) A modification embedding a regenerator, where the rejected heat Q_{41} is stored in the regenerator and then transferred back to the gas as Q_{23} , is proposed. When this modification is applied to the cycle, prove that the thermal efficiency of this modified cycle is the same as that of the Carnot cycle.
- (7) Although the modified cycle has not been practically realized, we can approximately realize the cycle with some improvements on the Brayton cycle (an ideal cycle for gas-turbine engine). How do you improve the Brayton cycle?

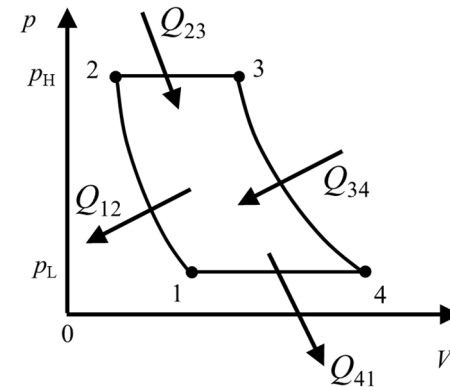


Fig. 1 $p-V$ diagram

(II) Both sides of the flat plate having thickness $2L$ (m), thermal conductivity k (W/(m·K)), density ρ (kg/m³), specific heat c (J/(kg·K)), and uniform volumetric heat generation of H (W/m³) are exposed to a fluid of temperature T_∞ (K). Under steady state conditions, the temperature distribution inside the flat plate is $T(x) = \alpha + \beta x + \gamma x^2$ (K), where $\alpha > 0$, $\beta < 0$, and $\gamma < 0$. Answer the following questions using L , k , ρ , c , T_∞ , x , α , β , and γ as necessary. (25 points)

- (1) Find the heat flux distribution $q(x)$ in the flat plate. And find the position x where the temperature gradient becomes zero.
- (2) Find the heat fluxes $q(x)$ at $x = L$ and $x = -L$.
- (3) Find the volumetric heat generation of H (W/m³).
- (4) Find the heat transfer coefficients h at $x = L$ and $x = -L$.
- (5) Find the temperature of the flat plate after enough time has elapsed after the heat generation inside the flat plate has stopped. How much heat per unit area (J/m²) must be removed after the heating is stopped before reaching the steady state?

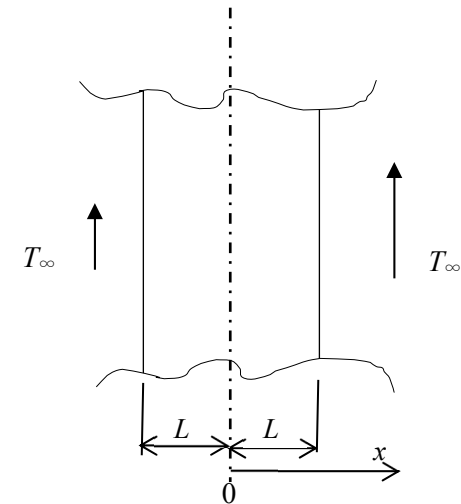


Fig.2